

Effect of Stress Distribution on Transverse and Longitudinal Framing in Cargo Ship using FEA Method



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Abstract: A cargo ship's hull is made up of bent plates that are not stiffened. The hull can be strengthened by the structural member of the transverse or longitudinal framing system. Frames are strengths member that acts as integral parts of the ship girder when the ship exposed to the longitudinal and transverse stresses. One of the stresses is coming from the load by the container that the cargo ships carry. This research was conducted to analyse how the load does from the container affecting the total deformation and the stress distribution on the transverse framing system model. The analysis of model was using finite element analysis method. Finite element analysis is a numerical technique for solving engineering issues with complex loadings, geometries and material properties. The simulation results of equivalent (von-Mises) stress and total deformation will be compared with the Germanisher Lloyd rule. The result of total deformation and equivalent (von-Mises) stress must not exceed the value stated by Germanischer Lloyd rule. If the value of stress exceeds, it is considered a failure. All the results are not exceed the limit and is acceptable.

Keywords: transverse, longitudinal, load, stress, deformation

I. INTRODUCTION

A cargo ship is a ship that transport materials and heavy goods from one port to another port. In the shipping industry, a cargo ship is the best way of transfer of materials,

transportation and any sort of business item because it is very safe in the handling of materials and is quite cheap [1]. A ship's hull is basically made up of bent plates welded together and not stiffened. So, the plates are stiffened by adding stiffeners to it. There are two ways to stiffen a ship which is transverse stiffening and longitudinal stiffening [2].

It is important to design a cargo ship that can withstand the weight of cargo. The designer plays a vital role to calculate the load distribution in a cargo ship. With the technology of Finite Element Analysis (FEA), structural failure of the ship can be studied. FEA is used in new product design and existing product refinement. Finite element analysis capable to verify a proposed design to the user's specifications before construction or manufacturing. Finite element analysis is a computational technique used for solving engineering problems having complex geometries that are subjected to general boundary conditions [3].

Finite element software is used to simulate of all disciplines of fluid dynamics, physics, vibration, heat transfer, electromagnetic and structural for engineers. Before the prototypes of the product are manufacturing, The software can simulate tests or working conditions and enables to test the product in the virtual environment [4].

In cargo ship, the load is applied as pressure. A model of a container can be simulated using finite element software. So, the linear static analysis will be carried out. The von-Mises stress result at structural member can be compared with the actual yield value in order to check whether the values are within the limit or exceeds the limit [5]. The load effect analysis considers the responses such as the bending moment, deflection and stress caused by all types of loading distribution on the structural modeling [6].

The objectives of this research are to develop the model of one section of the cargo ship transverse and longitudinal framing system in by using modeling software work with following the DNVGL rules. The second objectives is to analyze the deformation and stress distribution on transverse and longitudinal framing system of the model by using finite element software whether the framing system pass the criterion of DNV-GL in case of the deformation and maximum (von-mises) stress.

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II. LITERATURE REVIEW

Cargo Ship

In mercantile law, the term cargo applied to a vessel that transports goods, materials and merchandise conveyed in a ship to be transported from one port to another. Technically, the term designates to carry goods only but somehow, it applied in such a sense as to include passengers. Cargo ships are characterized under different categories based on their capacity partly by size, dimension and weight [7]. According to [8], this cargo ships are segregates into the specification namely liners or tramp ships. The liners are ships that travel on routes at fixed scheduled intervals between specified ports while. The ships that travel only when offered to bring the cargo to any port at a fixed contractual price called tramp ships. According to shipping and freight resource, there are few different kinds of cargo ships namely as;

Table. 1 Types of cargo

Type of cargo	Function
Bulk carrier	- Which transport ores or other dry cargoes in bulk
Container ship	- Transport for handling shipments in standardized containers (20', 40' and 45')
Passenger ships	- Cruise ships, and ferries
Tanker	- Transport liquid cargo like petroleum products (oil based fuels, and natural gas).
Ro-ro vessel	- Transport of wheeled cargo like cars, buses, trucks, excavators.
General (multipurpose) cargo vessels	- A versatile which carry all kind of goods - making them very popular vessels for cargo shipping.
LNG carriers	- Transport of natural gas that has been liquefied
Crude carriers	- Transport crude oil; VLCC (Very large Crude Carriers) and ULCC (Ultra large Crude Carriers)

The Ship Hull

The hull structure design is considered as a very important part in shipyard management. It also known as foundation of the ship, because it is designed to withstands extremely harsh weather conditions due to high durability and resistance [9].

But the hull of ships structure is push down if it is loaded with cargo. According to [10], the large point loads may occur in the structure if the ships structure, equipment, and cargo are distributed unevenly. It also can cause bending forces on the hull if they are distributed differently than the distribution of buoyancy from displaced water.

- The hull also need to bear stresses from;
- The water pressure from surrounding presses on hull
- The ship weight (consists of cargo, component and \hull) heading right toward the hull.
- The waves run onto the hull while wind blows against it.

Transverse Framing System

On large ship, currently use the transverse framing system and the longitudinal framing system. The transverse framing system provides for continuous transverse frames with the widely spaced longitudinal intercostal between them [11]. Transverse frames are attached to the keel and extend from

the keel outward around the turn of the bilge and up to the edge of the main deck. They are closely spaced along the length of the ship, and they define the form of the ship [11].

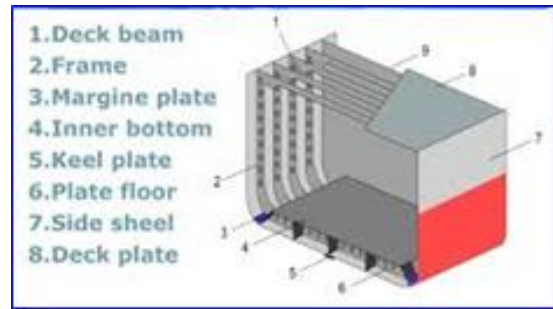


Fig. 1 Transverse framing member

Longitudinal Framing System

Longitudinal framing and is characterized by a plurality of longitudinal girders 7 running parallel to the longitudinal axis of the vessel. These girders are spaced along the sides and bottom of the vessel. Again, the longitudinal bulkheads 9 are mounted from the floor of the vessel vertically throughout a major portion of the hull. A plurality of longitudinal stiffeners or stringers 11 is mounted along the sides of the hull and along the longitudinal bulkheads. These stringers are also preferably in the horizontal plane and run parallel to the longitudinal axis of the vessel. To add additional transverse rigidity to the vessel, a number of transverse webs 13 are mounted within the hull at selected spaced locations.

The increasing size of cargo and the naval vessel through the 20th century, therefore, led to the general acceptance of combined longitudinal deep frame scantlings. A cross-section of typical longitudinal stiffening system is shown in Figure 2. The longitudinal are relatively small and closely spaced, the spacing is regular and the number of stiffeners is rationalized to meet local and global load requirements. Deep longitudinal girders are located at widely spaced intervals across the panel. The transverse web frames and floors run orthogonally to the longitudinal and are usually deeper but more widely spaced [12].

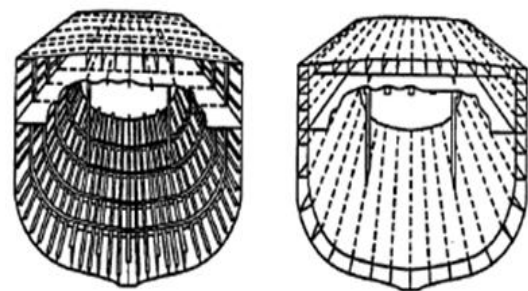


Fig. 2 Example conventional longitudinal framed (left) and advanced double hull (right) midship section arrangements

Longitudinal Framing vs Transverse Framing System

Table. 2 Difference between longitudinal framing and transverse framing

Transverse Framing	Longitudinal Framing
There is no influence to the ship’s resistance to longitudinal bending.	Longitudinal framing system is more resistant to buckling when subjected to compressive stresses while bending in the seaway.
Not optimum from the standpoint of structural efficiency. Transverse framing required strength for the least weight.	Longitudinal is higher structural efficiency. This is because most of the plating stiffeners are in the aft and fore direction.

DNVGL Rules for Classification and Construction

It is a must when designing a ship structure to follow the regulations that have been standardized by the classification society or AICS. In this research, the rules that been used is Germanischer Lloyd which is also known as GL rules.

These rules help us to determine the thickness of designing the model to be analyzed in the research. Every detail and the thickness of every part of the vessel can be determined.

The classification society also provides the arrangement guideline of the containers on the deck of the cargo ship. Based on the rules [13] it stated that if containers are stowed with linear seating in several layers, the total weight of the containers above the first layer shall not exceed the following values:

- 0.8G for 40ft containers and
- 1.0G for 20ft containers where G is the container’s maximum gross weight

III. METHODOLOGY

Gathering Data

Through web sources, the general arrangement for cargo ship is obtained. In this project, only one hull section that being used for simulation. Therefore, only that hull section has being studied from the general arrangement of the cargo ship. Hence, all the data such as its dimensions, type of structures and number of structures consists by the section from the ship that need to be collected before developing the model had been well analyzed as shown in Table 3 and Table 4.

Table. 3 Details for the cargo ship

Item	Dimension
LOA	92.5 m
Depth	6.7 m
Breadth	12.4 m
Draught	5.3 m

Table. 4 Details for Hull Structure in the model

Hull Part	No. of Parts
Hull	1
Deck Plate	1
Margin Plate	1
Bulkhead	2

Table. 5 Details for Transverse framing

Transverse Part	No. of Parts
Transverse Frame	17
Deck Beam	17

Table. 6 Details for Longitudinal framing

Longitudinal Part	No. of Parts
Longitudinal Frame	17 at bottom deck
	10 on each side shell
	17 at bottom hull

Developing The Model

The calculation to find the thickness of each plate in transverse framing system model is according to Germanischer Lloyd Classification.

▪ **Hull Section**

- The dimension of hull section;
- Length of section = 13460 mm
- Thickness = 113 mm
- Width of section = 12400 mm
- Height of section = 6700 mm
- Radius of the curve = 1500 mm

The thickness of each plate in transverse and longitudinal framing system is calculated according to Germanischer Lloyd Classification.

Table. 7 Thickness for each part in transverse frame

Part	Thickness (mm)
Deck Plate	7.74
Margin Plate	8.16
Keel Plate	8.16
Bulkhead	7.00
Transverse Frame	8.00
Deck beam	8.00

Table. 8 Thickness for each part in longitudinal frame

Part	Thickness (mm)
Deck Plate	7.74
Margin Plate	8.16
Keel Plate	8.16
Bulkhead	7.00
Longitudinal frame	8.00
Height (200mm)	
Width (150 mm)	

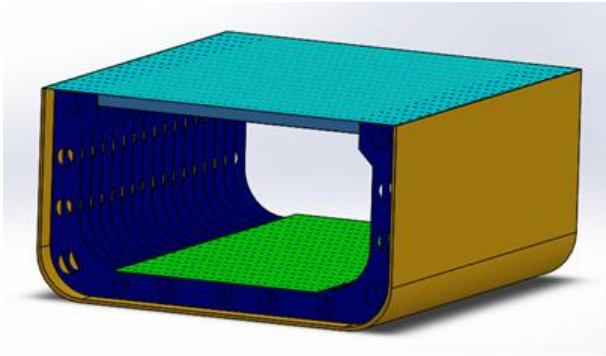


Fig. 3 Assemble model for Transverse Frame

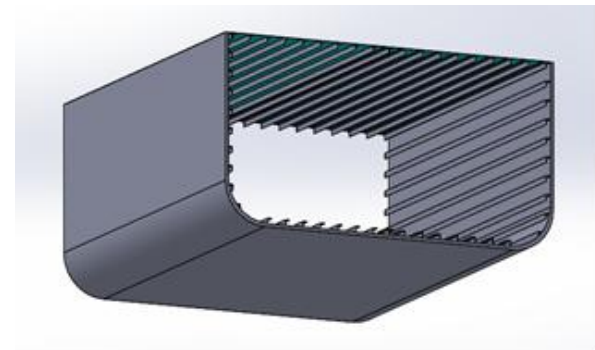


Fig. 4 Assemble model for Longitudinal Frame

Container Arrangement on the Model

Based on the DNVGL rules it has been stated that the cargo ship with this particular length can hold up to three tier of container on the deck [14]. Therefore, the simulation tested from one tier to three tiers of container and their load distribution. The deformation and the maximum stress will be analyzed. Figure 12 and 13 showed the arrangement of container in the cargo hold.

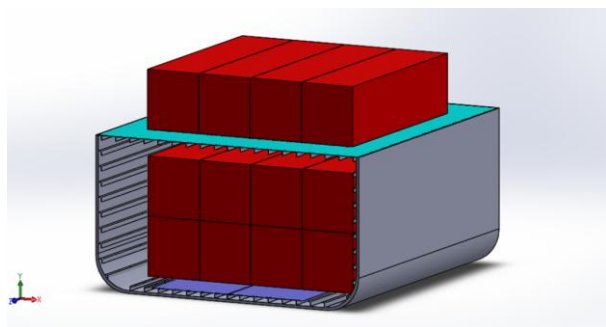


Fig. 5 Container arrangement in finite element software

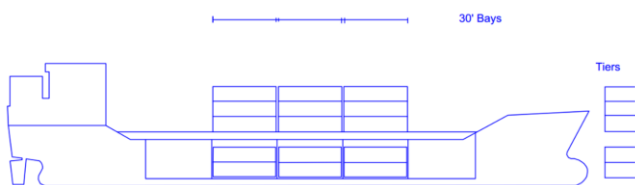


Fig. 6 Actual container arrangement for three tier actual in finite element software

Table. 9 Load Distributions

Maximum permitted gross weight of 1 container		25,400 kg	
Upper Deck			
Tier	No of containers	Weight (kg)	Force (N)
1	4	101,600	996,355
2	8	203,200	1,992,711
3	12	304,800	2,989,066
Inner Bottom			
Tier	No of containers	Weight (kg)	Force (N)
2	8	203,200	1,992,711

DNV GL Rule

Table. 10 Load Distributions

Grade	Yield strength R_{eH} [N/mm ² min.]	Tensile strength R_m [N/mm ²]	Elongation ¹ A (at $L_0 = 5.65 \cdot \sqrt{S_0}$) [%] min.	Test temp. [°C]	Notched bar impact energy					
					KV [J] min.					
					$t \leq 50$ [mm]		$50 < t \leq 70$ [mm]		$70 < t \leq 150$ [mm]	
	long.	transv.	long.	transv.	long.	transv.				
GL-A32 GL-D32 GL-E32 GL-F32	315	440-570 ²	22	0 -20 -40 -60	31	22	38	26	46	31
GL-A36 GL-D36 GL-E36 GL-F36	355	490-630 ²	21	0 -20 -40 -60	34	24	41	27	50	34
GL-A40 GL-D40 GL-E40 GL-F40	390	510-660 ²	20	0 -20 -40 -60	41	27	46	31	55	37

t = thickness of product [mm]

In this project, the steel use is grade GL-A36. So, the yield strength must not less than 355 N/mm² and the tensile strength not exceed 630 N/mm². The maximum elongation based on this rule is 21mm.

Convergence Study

The next step in simulation preparation is to determine the most suitable meshing size for the analysis. This is to get the most accurate to the actual result in real situation. The meshing is the process of dividing the whole model to into number of elements so that it will analyzed more accurately by computing all the part of the model perfectly. By doing this, it will give more realistic result.

In order to determine the most suitable element size for meshing, the convergence study should be made. The convergence study is the process which several element sizes was used to run the analysis and then a graph of element size versus maximum stress was plotted. The most suitable size was determined by choosing the starting point of when the graph saturated. The smaller the element size of the model will give us more accurate result. However, the smaller the element size it will consume more time and need a powerful computer to run the analysis. Therefore, the convergence study help to save time and cost while giving a reasonable result.

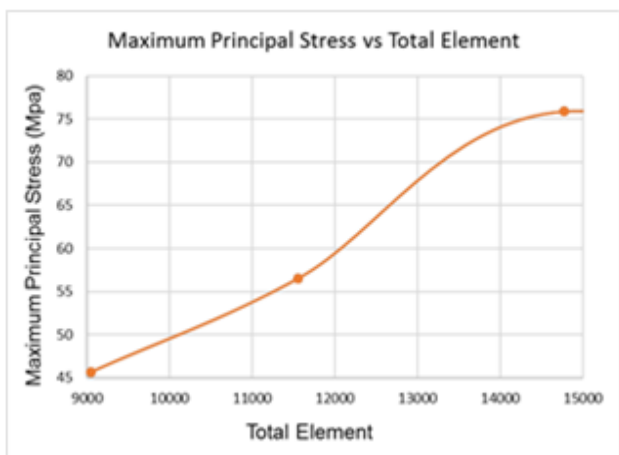


Fig. 7 Convergence Study Graph

The element size is varied from 0.2, 0.25, 0.3, 0.35 and 0.4. The lower the element size, the total element for meshing will be increased. From the graph, it showed that the graph is constant after the total element increased to 15000. The element size for this total element was 0.3. Therefore it can be said that 0.3 element size is the most suitable element size for this project’s simulation. The factor is 0.3 was the starting point for the graph to be saturated and no significant changes on the graph. Therefore, it is the most suitable and more realistic to be run in the analysis.

IV. RESULT AND DISCUSSION

Transverse Frame

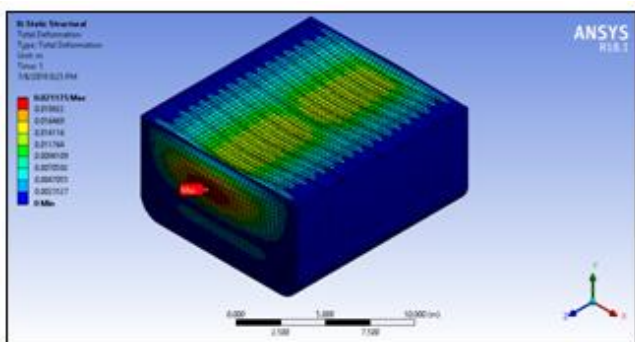


Fig. 8 Total Deformation for Overall model

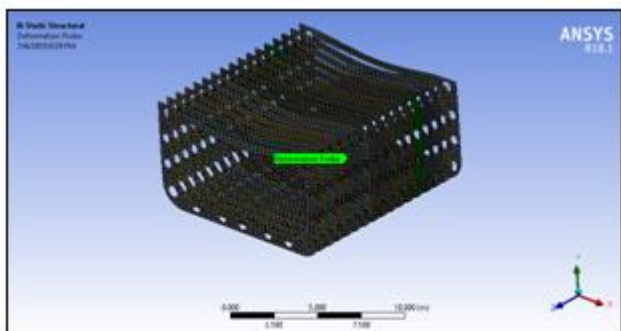


Fig. 9 Total Deformation for Individual Frame

Table. 11 Maximum Deformation & Equivalent Stress of the model (Transverse)

	Deformation (mm) & Part	Equivalent stress (MPa) & Part
1 tier	5.954, Bulkhead	104.7, Bulkhead
2 tier	9.282, Bulkhead	145.05, Deck Beam
3 tier	15.204, Bulkhead	213.92, Deck Beam

Based on the Table 11 above, showed the result of simulation for each different value of load so that it was easier to compare with the actual value for the stress stated by DNVGL rules. The result obtained from the analysis showed that none of the maximum deformation of the part neither the equivalent stress bear by the model reach the value from the rules.

Longitudinal Frame

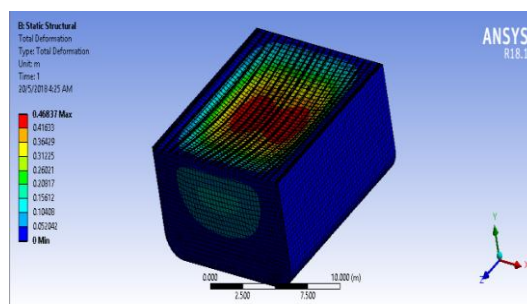


Fig. 10 Total Deformation for Overall model

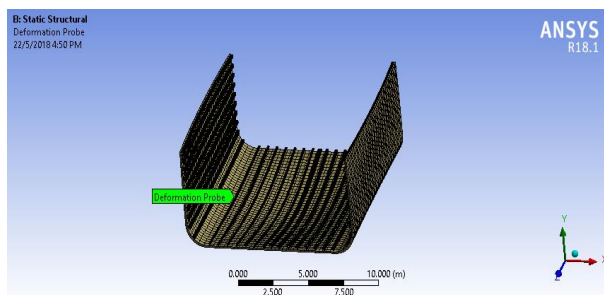


Fig. 11 Total Deformation for Individual Frame

Table. 12 Maximum Deformation & Equivalent Stress of the model at Longitudinal Beam

	Deformation (mm)	Equivalent stress (MPa)
1 tier	468.37	879
2 tier	702.67	1320
3 tier	936.96	1760

From the Table 12, the longitudinal beam experienced the deformation and maximum stress. Both of the components are the longitudinal frame which the maximum stress at the deck longitudinal frame. The result obtained from the analysis showed that all the deformation and equivalent stress is exceed the DNVGL rules. Based on DNVGL rules, the maximum high tensile strength for steel and maximum deformation are 630MPa and 21mm respectively.

V. CONCLUSION AND RECOMMENDATION

It can be conclude that, for transverse framing system all the result are acceptable based on the DNVGL rules and the transverse framing system surely can hold the load from the container. However, for longitudinal framing system the result obtained from this analysis concludes that this model required more support other than just longitudinal framing system. This is because the stress obtained already exceed the tensile strength from the rules and achieve failure. The thickness of the structural components also can give a great impact to increase the strength of the structure.

Longitudinal strength of ships is a critical factor which affects both the design and operation of the ship. Careful planning and supporting calculations during the design stage should be carried out to ascertain the vessel's suitability from the strength point of view.

There are some recommendation to increase or improve the capability and strength of the structures of hull section. This will help the cargo ship to be more sufficient during operations and also increase the lifespan of the cargo ship by reducing any major damages towards the structures.

Therefore, it is recommended to use the thicker material especially on the hull section or add more supportive structures due to increase the capability of the hull structure to withstand the loads hence will reduce the total deformation and also stress.

Last but not least, choosing the right type of materials for the structures based on the ship's operations purposed also influenced the strength of the ship. This is because different materials consist of different yield and tensile strength.

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REFERENCES

1. G. O. Young, "Synthetic structure of industrial plastics (Book style with paper title and editor)," in *Plastics*, 2nd ed. vol. 3, J. Peters, Ed. New York: McGraw-Hill, 1964, pp. 15–64.
2. W.-K. Chen, *Linear Networks and Systems* (Book style). Belmont, CA: Wadsworth, 1993, pp. 123–135.
3. H. Poor, *An Introduction to Signal Detection and Estimation*. New York: Springer-Verlag, 1985, ch. 4.
4. B. Smith, "An approach to graphs of linear forms (Unpublished work style)," unpublished.
5. E. H. Miller, "A note on reflector arrays (Periodical style—Accepted for publication)," *IEEE Trans. Antennas Propagat.*, to be published.

6. J. Wang, "Fundamentals of erbium-doped fiber amplifiers arrays (Periodical style—Submitted for publication)," *IEEE J. Quantum Electron.*, submitted for publication.
7. C. J. Kaufman, Rocky Mountain Research Lab., Boulder, CO, private communication, May 1995.
8. Y. Yorozu, M. Hirano, K. Oka, and Y. Tagawa, "Electron spectroscopy studies on magneto-optical media and plastic substrate interfaces(Translation Journals style)," *IEEE Transl. J. Magn.Jpn.*, vol. 2, Aug. 1987, pp. 740–741 [Dig. 9th Annu. Conf. Magnetics Japan, 1982, p. 301].
9. M. Young, *The Technical Writers Handbook*. Mill Valley, CA: University Science, 1989.
10. (Basic Book/Monograph Online Sources) J. K. Author. (year, month, day). Title (edition) [Type of medium]. Volume(issue). Available: [http://www.\(URL\)](http://www.(URL))
11. J. Jones. (1991, May 10). *Networks* (2nd ed.) [Online]. Available: <http://www.atm.com>
12. (Journal Online Sources style) K. Author. (year, month). Title. Journal [Type of medium]. Volume(issue), paging if given. Available: [http://www.\(URL\)](http://www.(URL))

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